

A mean stress model of fatigue life of metal materials under multiaxial loading

Abstract

Due to the complexity of multiaxial fatigue damage of metal materials, up to date, it is still a challenging task to establish a multiaxial fatigue model with influence of different mean stress. In this paper, a linear mean stress model is presented on basis of the multiaxial model of fatigue life of metal materials by Liu and Yan. By using the experimental data of fatigue life of metal materials reported in the literature, the model is systematically validated.

Keywords: multiaxial fatigue, mean stress, metallic materials

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Introduction

Many critical mechanical components experience multiaxial cyclic loading during their service life, such as railroad wheels, crankshafts, axles, and turbine blades, etc. Different from the uniaxial fatigue problem, the multiaxial fatigue problem is more complex due to the complex stress states. In recent decades, a significant amount of research has been devoted to acquire a better understanding of the failure mechanisms under multiaxial loading, including theoretical model study (e.g., a stress invariant method,^{1,5-10} and a critical plane method,¹¹⁻¹³ fatigue tests of metallic materials under multiaxial loading (e.g., fatigue test for engineering steels by Gough³ and fatigue of wrought high-tensile alloy steel by Frith.⁴ Due to the complexity of multiaxial fatigue damage of metal materials, up to date, it is still a challenging task to establish a multiaxial fatigue model with influence of different mean stress. In this paper, a linear mean stress model is presented on basis of the multiaxial model of fatigue life of metal materials by Liu & Yan^{1,2} By using the experimental data of fatigue life of metal materials reported in the literature, the model is systematically validated.

A multiaxial mean stress model

In this section, the Marin's mean stress model proposed by Liu and Yan¹ is first described briefly. Then a linear mean stress model is given. After the reader reads this paper, the reason that the linear mean stress model is presented by the author is naturally understood.

A Marin's mean stress model

A model for fatigue life prediction under multiaxial stress states can be expressed mathematically as follows:

$$F(\sigma_{ea}, \sigma_{em}, \rho, N, c_1, c_2, \dots) = 0 \quad (1)$$

where σ_e is a mechanical parameter which is a measure of stress states in multiaxial loading, and here, the von Mises equivalent stress is adopted; σ_{ea} and σ_{em} are the amplitude and the mean value of the equivalent stress, respectively. Multiaxial parameter ρ is defined as follows:

$$\rho = \frac{\sigma_{11,a}}{\sigma_{e,a}} \quad (2)$$

where $\sigma_{11,a}$ is the amplitude of the first invariant of stress tensor. It is evident that, for the axial and shear fatigue loading, the value of the multiaxial parameter ρ is equal to 1 and 0, respectively. c_1, c_2, \dots in the Eq. (1) are material coefficients, which are varied with the multiaxial parameter ρ . Based on the previous wide investigations (e.g., Tao and Xia¹⁴ and Marin's general equation¹⁵) of engineering rules suitable for predicting the mean stress effect under the bending loading, it is assumed here that the model of multiaxial fatigue life has the following form:

$$\log \sigma_{ea} (1 - \alpha \frac{\sigma_{em}}{\sigma_{ea}})^{n_\rho} = A_\rho \log N + C_\rho \quad (3)$$

where $A_\rho, C_\rho, \alpha_\rho, n_\rho$ are material parameters dependent on the multiaxial parameter ρ . For the axial and shear fatigue loading, Eq. (3) can be simplified as, respectively:

$$\log \sigma_a (1 - \alpha_1 \frac{\sigma_m}{\sigma_a})^{n_1} = A_1 \log N + C_1 \quad (4)$$

And

$$\log \sqrt{3} \tau_a (1 - \alpha_0 \frac{\tau_m}{\tau_a})^{n_0} = A_0 \log N + C_0 \quad (5)$$

In the absence of mean stress, the Eq. (3) is simplified as:

$$\log \sigma_{ea} = A_\rho \log N + C_\rho \quad (6)$$

Thus under the axial and shear fatigue loading, the Eqs (4) and (5) can be written as respectively:

$$\log \sigma_a = A_1 \log N + C_1 \quad (7)$$

And

$$\log \sqrt{3} \tau_a = A_0 \log N + C_0 \quad (8)$$

Eq. (7) is the well known *S-N* curve equation. Eq. (8) is the variant form of the *S-N* curve equation under the shear fatigue condition, which can be written as:

$$\log \tau_a = A_0' \log N + C_0' \tag{9}$$

in which A_0' and C_0' are related with A_0 and C_0 through the following relationships:

$$A_0' = A_0, C_0' = C_0 - \sqrt{3} \tag{10}$$

In view of the complexity of fatigue life analysis in the multiaxial stress states, and also taking into account that the existing literature has accumulated a large number of fatigue experimental data under the axial and shear loading, from the point of application, it is assumed that the material parameters in Eq. (3) can be obtained by interpolating the material parameters in Eqs (4) and (5), i.e.:

$$A_\rho = A_1 \cdot \rho + A_0 \cdot (1 - \rho) \tag{11}$$

$$C_\rho = C_1 \cdot \rho + C_0 \cdot (1 - \rho) \tag{12}$$

$$\alpha_\rho = \alpha_1 \cdot \rho + \alpha_0 \cdot (1 - \rho) \tag{13}$$

$$n_\rho = n_1 \cdot \rho + n_0 \cdot (1 - \rho) \tag{14}$$

In this way, multiaxial fatigue life can be directly estimated from the stress invariant parameter and the multiaxial S-N curve with the attention on mean stress effect.

A linear mean stress model

According to the Marin’s mean stress model of multiaxial loading, i.e., Eq.(3), a linear mean stress model presented in this paper is as follows:

$$\log \sigma_{ca} (\beta_\rho - \gamma_\rho \frac{\sigma_{em}}{\sigma_{e0}}) = A_\rho \log N + C_\rho \tag{15}$$

where

$$\sigma_{e0} = \sigma_0^\rho (\sqrt{3}\tau_0)^{(1-\rho)} \tag{16}$$

in which σ_0 and τ_0 are, respectively, a tensile fatigue limit and a shear fatigue limit. Under the axial and shear fatigue loading, Eq (15) can be written as

$$\log \sigma_a (\beta_1 - \gamma_1 \frac{\sigma_m}{\sigma_0}) = A_1 \log N + C_1 \tag{17}$$

and

$$\log \sqrt{3}\tau_a (\beta_0 - \gamma_0 \frac{\tau_m}{\tau_0}) = A_0 \log N + C_0 \tag{18}$$

in which $\beta_1, \gamma_1, \beta_0$ and γ_0 are material constants determined by fitting experimental data of fatigue life.

By the way, material parameters and in Eq (15) can be determined by using the interpolation formulas similar to formulas (13) to (14). Eq (16) is a multiaxial fatigue limit prediction equation recently proposed by Liu & Yan²

Experimental verifications and discussions

Based on the experimental data of fatigue life of metallic materials from literature, in this section, fatigue life analysis will be carried out by using fatigue life prediction equations with the linear mean stress effect and the Marin’s mean stress effect. The comparison of two results will be given. In order to quantitatively evaluate the accuracy of the fatigue life prediction, the following error indexes are defined:

$$ER1 = \frac{(N_{cal} - N_{exp}) \times 100}{N_{exp}} \tag{19}$$

$$ER2 = \frac{(N_{cal}' - N_{exp}) \times 100}{N_{exp}} \tag{20}$$

$$MER1 = \frac{1}{n} \sum_{i=1}^n (ER1)_i \tag{21}$$

$$MER2 = \frac{1}{n} \sum_{i=1}^n (ER2)_i \tag{22}$$

where n is the number of experimental cases, N_{exp} is experimental fatigue life; N_{cal} and N_{cal}' are computed fatigue lives by the Marin’s mean stress model and the linear mean stress model, respectively. In this section, the fatigue life prediction of three metallic materials, Cast iron, 18G2A steel, and S355J0 alloy steel under uniaxial and multiaxial loading are carried out. For the sake of clear discussions, they are described in the forms of examples, respectively. The material constants determined by using the experimental data of fatigue life of these materials are listed in Table 1 & Table 2. Note here material constants, σ_0 and τ_0 , can be evaluated respectively by means of Eqs (7) and (8). For example, for 18G2A steel, let $N_f = 2000000$ (cycles), the following two values can be obtained.

$$\sigma_0 = 286 \text{ (MPa)}, \tau_0 = 188 \text{ (MPa)}$$

Table 1 Material constants and R-square of fitted S-N equation under the axial fatigue loading

Materials	A_1	C_1	R-square	α_1	n_1	β_1	γ_1
Cast iron	-0.140	2.858	0.840	-0.499	1.210	1.663	-0.039
18G2A steel	-0.131	3.286	0.932	-12.13	0.139	1.161	-0.279
S355J0 alloy steel	-0.177	3.535	0.893	-4.316	0.215	1.059	-0.343

Table 2 Material constants and R-square of fitted S-N equation under the shear fatigue loading

Materials	C_0	C_0	R-square	α_0	β_0	β_0	γ_0
Cast iron	-0.074	2.887	0.965	-0.604	0.700	1.357	-0.044
18G2A steel	-0.074	2.984	0.884	-27.233	0.133	1.223	-0.379
S355J0 alloy steel	-0.076	2.994	0.670	-138.02	0.079	1.290	-0.251

Example 1: Cast iron

For cast iron reported by Berto, Lazzarin & Tovo¹⁶ under zero mean stress, predicted results of fatigue life are given in Table 3. Error ranges of fatigue life are, respectively, [-47, 80](%), [-41, 66](%) and [-43,28](%), for the axial, shear and multiaxial fatigue loading, which shows that the fatigue lives predicted by means of Eqs (6), (7) and (8) are very satisfactory. Note here that there is an early failure case with a relative error 746%, see Table 3, which should be deleted. Under the axial fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER2$, are, respectively, [-28,44] (%) and [-29,45] (%), see Table 4, which illustrates that fatigue lives predicted by means of the linear mean stress model, i.e., Eq (17), almost are same as those by means of the Marin’s mean stress model, i.e., Eq (4). Under the shear fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER2$, are, respectively, [-34,30] (%) and [-34,23] (%), see Table 4, from which it can be seen that accuracy of fatigue lives predicted by using the two mean stress models is very high, and that the fatigue lives predicted by the linear mean stress model, i.e., Eq. (18), are a little better than those by the Marin’s mean stress model, i.e., Eq. (5). Under the multiaxial fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER2$, are, respectively, [-72, 30] (%) and [-62, -16] (%), see Table 4, which shows that that accuracy of fatigue lives predicted by using the two mean stress models is satisfactory (Table 3).

Example 2: 18G2A steel

For 18G2A steel reported by Gasiak & Pawliczek¹⁷, under zero mean stress, predicted results of fatigue life are given in Table 5. Error ranges of fatigue life are, respectively, [-28, 105] (%), [-34, 81] (%) and [-54,-13] (%), for the axial, shear and multiaxial fatigue loading, which shows that the fatigue lives predicted by means of Eqs (6), (7) and (8) are very satisfactory. Under the axial fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER1$, are, respectively, [-45,92] (%) and [-40,92] (%), see Table 6, which illustrates that fatigue lives predicted by the linear mean stress model, i.e., Eq (17) almost are same as those by means of the Marin’s mean stress model, i.e., Eq (4). Under the shear fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER1$, are, respectively, [-46,53](%) and [-54,46] (%), see Table 6, from which it can be seen that accuracy of fatigue lives predicted by using the Marin’s mean stress model, i.e., Eq (5) almost is same as that by means of the linear mean stress model, i.e., Eq (18). Under the multiaxial fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER2$, are, respectively, [-49,172](%) and [-42,170](%), see Table 7, which shows that the fatigue lives predicted by using the linear mean stress model, i.e., Eq (15), are a little better than those by the Marin’s mean stress model, i.e., Eq (3) (Table 4 -Table 7).

Example 3: S355J0 alloy steel

For S355J0 alloy steel reported by Gasiak & Pawliczek¹⁸ under

zero mean stress, predicted results of fatigue life are given in Table 8. Error ranges of fatigue life are, respectively, [-25,17] (%) and [-33,81] (%), for the axial and shear fatigue loading, which shows that the fatigue lives predicted by means of Eqs (7) and (8) are very accurate. Under multiaxial fatigue loading, error range of fatigue life, [-68,-13] (%), is satisfactory. Under the axial fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER2$, are, respectively, [-57, 45] (%) and [-51, 39] (%), see Table 9, which illustrates that accuracy of fatigue lives predicted by using the two mean stress models is very high, and that the fatigue lives predicted by the linear mean stress model, i.e., Eq (17), are a little better than those by the Marin’s mean stress model, i.e., Eq (4). Under the shear fatigue loading with mean stress, the error ranges of fatigue life, $ER1$ and $ER2$, are, respectively, [-42, 79] (%) and [-48, 86] (%), see Table 9, from which it can be seen that accuracy of fatigue life predicted by using the two mean stress models is very high, and that the fatigue lives predicted by the Marin’s mean stress model, i.e., Eq (5), are a little better than those by the linear mean stress model, i.e., Eq (18). Under the multiaxial fatigue loading with mean stress, the error range and the mean error of fatigue lives predicted by using the linear mean stress model, i.e., Eq (15), are, respectively, [-38,99] (%) and 8%, see Table 9, which shows that that accuracy of fatigue lives predicted by using the linear mean stress model is very high. However, the error range and the mean error of fatigue lives predicted by means of the Marin’s mean stress model, i.e., Eq (3), are, respectively, [-89,-68] (%) and -84%, see Table 9, which shows that the predicted by the Marin’s mean stress model is very poor. The comparison of the two results is shown in Figure 1. What is the reason that, under the multiaxial fatigue loading with mean stress, the accuracy of fatigue lives of S355J0 alloy steel predicted by the Marin’s mean stress model is very poor? For this, the author observes the material constants listed in Table 1 & Table 2 and finds that, for S355J0 alloy steel, α_0 is much less than α_0 , ($\alpha_1 / \alpha_0 = 0.03127$, see Table 10), which is the reason that the accuracy of fatigue lives predicted by using the Marin’s mean stress model, i.e., Eq. (3), is very poor. In order to further illustrate it, in the following form of Marin’s mean stress model:

$$\log \sigma_{ea} \left(1 - \alpha \frac{\sigma_{em}}{\sigma_{ea}}\right)^n = A_\rho \log N + C_\rho \tag{23}$$

by letting $\alpha = \alpha_1, \alpha = \alpha_0$ and $\alpha = \alpha_0, n = n_0$, fatigue lives predicted are listed in 11, in which the error ranges are, respectively, [-60,16](%) and [-37,188] (%), and the mean errors are, respectively, -21(%) and 38(%) (Table 12). Comparison of detail errors are given in Table 12, from which it can be seen that the accuracy of fatigue lives predicted by using the linear mean stress model, i.e., Eq (15), is much better than that by using the Marin’s mean stress model, Eq (3).

Table 3 Comparison of experimental data and predicted results of fatigue life of Cast iron with zero mean stress

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp} (Cycles)	ER (%)
160	0	0	0	40,194	15
160	0	0	0	86,632	-47
130	0	0	0	159,000	28
130	0	0	0	193,903	5
108.5	0	0	0	408,667	80
108.5	0	0	0	1,091,520	-32
0	0	220	0	10,000	36
0	0	200	0	38,500	28
0	0	200	0	45,750	8
0	0	180	0	338,500	-39
0	0	180	0	350,000	-41
0	0	180	0	285,000	-28
0	0	160	0	980,000	3
0	0	140	0	3,698,000	66
0	0	140	0	5,055,500	21
140	0	140	0	16,400	-43
130	0	130	0	28,500	-34
120	0	120	0	46,800	-16
110	0	110	0	103,000	-14
100	0	100	0	229,000	-5
100	0	100	0	297,500	-27
90	0	90	0	68,500	746
90	0	90	0	520,000	12
90	0	90	0	602,000	-4
80	0	80	0	1,361,500	28
80	0	80	0	1,998,000	-13

Table 4 Comparison of experimental data and predicted results of fatigue life of Cast iron with non-zero mean stress

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp} (Cycles)	ER1 (%)	ER2 (%)
100	100	0	0	46,121	-13	-10
80	80	0	0	136,081	44	45
70	70	0	0	709,456	-28	-29
60	60	0	0	1,389,28	10	7
0	0	150	150	21,000	30	23
0	0	140	140	61,890	12	9
0	0	140	140	73,000	-5	-7
0	0	130	130	286,830	-34	-34
0	0	120	120	597,350	-6	-4
0	0	110	110	1,663,00	9	15
0	0	110	110	1,720,00	5	11
80	80	80	80	43,525	-5	-16
90	90	90	90	20,535	-54	-41
60	60	60	60	1,410,00	-4	-62
70	70	70	70	180,000	18	-29
70	70	70	70	173,000	23	-26
100	100	100	100	9000	-72	-49
60	60	60	60	1,040,00	30	-48

Table 5 Comparison of experimental data and predicted results of fatigue life of 18G2A steel with zero mean stress

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp}	ER (%)
399.6	0	0	0	128630	22
399.8	0	0	0	153881	2
399.8	0	0	0	168309	-7
364.1	0	0	0	327016	-3
365.3	0	0	0	375852	-17
367	0	0	0	415067	-28
275	0	0	0	1306495	105
275.4	0	0	0	2869140	-8
275.5	0	0	0	3364694	-21
0	0	223	0	224794	-10

Table Continued

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp}	ER (%)
0	0	222.6	0	278939	-26
0	0	222.7	0	310708	-34
0	0	195.7	0	637980	81
0	0	195.4	0	785106	50
0	0	196.2	0	874370	27
0	0	183.9	0	2296066	15
0	0	184	0	3095310	-15
0	0	183	0	4173870	-32
199.7	0	199.7	0	168497	-13
199.7	0	199.7	0	192588	-24
199.5	0	199.5	0	226543	-35
180.2	0	180.2	0	609639	-35
180.2	0	180.2	0	615487	-36
180	0	180	0	737954	-46
164.5	0	164.5	0	1303741	-27
164.6	0	164.6	0	1407201	-32
164.4	0	164.4	0	2101434	-54

Table 6 Comparison of experimental data and predicted results of fatigue life of 18G2A steel with non-zero mean stress (under uniaxial fatigue loading)

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp}	ER1 (%)	ER2 (%)
369.5	123.2	0	0	57172	-13	-24
369.6	123.2	0	0	65725	-24	-34
369.1	123	0	0	72614	-31	-40
298.3	99.4	0	0	131431	92	92
298.5	99.5	0	0	168583	49	49
298.5	99.5	0	0	178962	40	41
268.8	89.6	0	0	578316	-4	2
268.9	89.6	0	0	671481	-17	-12
268.9	89.6	0	0	712823	-22	-17
247.3	82.4	0	0	976665	7	19
247.1	82.4	0	0	1356769	-22	-14
290.2	290.2	0	0	95759	31	14
288.6	288.6	0	0	133071	-2	-14
265.7	265.7	0	0	157011	56	54
274.3	274.3	0	0	173047	11	5
255	255	0	0	241679	39	45
254.5	254.5	0	0	422196	-20	-16
235.3	235.3	0	0	887342	-31	-20
235.4	235.4	0	0	1126890	-45	-37
0	0	175	58.3	157758	-25	-35

Table Continued

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp}	ER1 (%)	ER2 (%)
0	0	174.5	58.2	229202	-46	-54
0	0	151.6	50.5	551271	46	47
0	0	151.3	50.4	640123	29	30
0	0	151.3	50.4	689753	20	21
0	0	138.4	46.1	2176479	25	37
0	0	138.4	46.1	3187921	-15	-6
0	0	138.2	138.2	356608	53	46
0	0	137.8	137.8	526773	8	4
0	0	135.4	135.4	710704	1	2
0	0	135.2	135.2	860208	-15	-14
0	0	131.4	131.4	1123272	-5	3
0	0	131.2	131.2	1501895	-27	-21

Table 7 Comparison of experimental data and predicted results of fatigue life of 18G2A steel with non-zero mean stress (under multiaxial fatigue loading)

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp}	ER1 (%)	ER2 (%)
146.5	48.8	146.5	48.8	87304	172	170
146.7	48.9	146.7	48.9	93327	151	149
146.5	48.8	146.5	48.8	121935	95	94
129.8	43.3	129.8	43.3	382788	100	117
129.8	43.3	129.8	43.3	417127	83	100
121.4	40.5	121.4	40.5	1408351	4	18
121.4	40.5	121.4	40.5	1505671	-3	11
118.5	39.5	118.5	39.5	2317350	-20	-8
118.3	39.4	118.3	39.4	3056739	-39	-29
139.4	139.4	139.4	139.4	115525	3	-15
139.4	139.4	139.4	139.4	138497	-14	-29
129.2	129.2	129.2	129.2	211846	17	11
129.5	129.5	129.5	129.5	233041	4	-1
123.2	123.2	123.2	123.2	257221	53	58
123.2	123.2	123.2	123.2	366179	7	11
120.9	120.9	120.9	120.9	681921	-31	-26
120.9	120.9	120.9	120.9	794444	-41	-36
118	118	118	118	1165727	-49	-43

Table 8 Comparison of experimental data and predicted results of fatigue life of S355J0 alloy steel with zero mean stress

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp} (Cycles)	ER (%)
400	0	0	0	153000	17
365	0	0	0	403000	-25
275	0	0	0	1295000	14
0	0	224	0	300000	-33
0	0	196	0	632000	81
0	0	184	0	3150000	-17
200	0	200	0	190000	-13
183	0	183	0	739000	-55
167	0	167	0	2117000	-68

Table 9 Comparison of experimental data and predicted results of fatigue life of S355J0 alloy steel with non-zero mean stress

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp} (Cycles)	ER1 (%)	ER2 (%)
247	82	0	0	1011000	-9	7
269	90	0	0	713000	-20	-11
368	123	0	0	72000	35	21
236	236	0	0	1083000	-57	-51
257	257	0	0	246000	17	19
269	269	0	0	153000	45	39
275	275	0	0	171000	15	6
290	290	0	0	131000	11	-6
0	0	139	46	1043000	79	86
0	0	151	50	670000	-6	-7
0	0	175	58	158000	-42	-48
0	0	133	133	1130000	-6	0
0	0	136	136	850000	-6	-4
0	0	139	139	525000	14	13
118	39	118	39	2356000	-89	-28
121	40	121	40	1516000	-86	-9
130	43	130	43	378000	-68	99
121	121	121	121	803000	-92	-38
124	124	124	124	366000	-86	8
130	130	130	130	210000	-84	21
140	140	140	140	117000	-84	8

Table 10 Comparison of material constants used to describe the effect of mean stress on fatigue life

Materials	Marin's mean stress model		Linear mean stress model	
	α_1 / α_0	n_1 / n_0	β_1 / β_0	γ_1 / γ_0
Cast iron	0.826159	1.726845	1.225497	0.88636
18G2A steel	0.445415	1.045113	0.949305	0.736148
S355J0 alloy steel	0.031271	2.697616	0.820739	1.366534

Table 11 Comparison of experimental data and predicted results of fatigue life of S355J0 alloy steel under multiaxial fatigue loading with non-zero mean stress

σ_a (MPa)	σ_m (MPa)	τ_a (MPa)	τ_m (MPa)	N_{exp}	$\alpha = \alpha_1, n = n_1$	$\alpha = \alpha_0, n = n_0$
					ERI (%)	ERI (%)
118	39	118	39	2356000	-60	-1
121	40	121	40	1516000	-49	26
130	43	130	43	378000	16	188
121	121	121	121	803000	-52	-37
124	124	124	124	366000	-12	15
130	130	130	130	210000	5	38
140	140	140	140	117000	5	38

Table 12 Error Indexes of different mean stress models under multiaxial fatigue loading (S355J0 alloy steel)

Marin's mean stress model				Linear mean stress model			
$\alpha = \alpha_1, n = n_1$		$\alpha = \alpha_0, n = n_0$		$\alpha = \alpha_\rho, n = n_\rho$			
Error range (%)	Mean error (%)	Error range (%)	Mean error (%)	Error range (%)	Mean error (%)	Error range (%)	Mean error (%)
[-60,16]	-21	[-37,188]	38	[-89,-68]	-84	[-38,99]	8

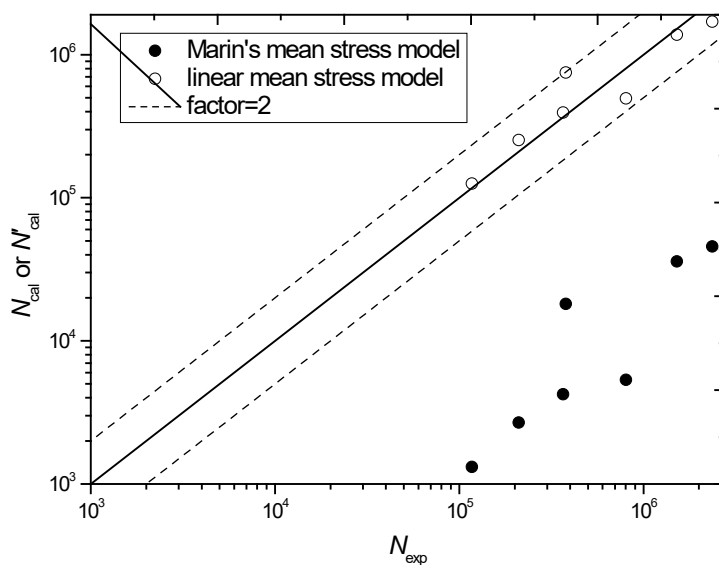


Figure 1 Comparison of fatigue lives predicted by the linear mean stress model and the Marin's mean stress model under multiaxial fatigue loading with non-zero mean stress.

Concluding remarks

On basis of the present investigation, it can be seen that:

- a) 1、 For some metal materials, for example, Cast Iron, an accuracy of fatigue lives predicted by means of the linear mean stress model and the Marin's mean stress model is very high and the two models almost have same accuracy.
- b) 2、 For S355J0 alloy steel, under the axial and pure shear fatigue loading, an accuracy of fatigue lives predicted by means of the linear mean stress model and the Marin's mean stress model is very high. But under the multiaxial fatigue loading, an accuracy of fatigue lives predicted by using the Marin's mean stress model is very poor, while on the contrary, fatigue lives predicted by means of the linear mean stress model are very satisfactory. On basis of the two points above, the author considers that the linear mean stress model presented in this paper is more suitable than the Marin's mean stress model in performing the fatigue life assessment. Up to date, as well known to us, the Marin's mean stress model has been widely used to perform the fatigue life assessment of metal materials with mean stress effect.

Why is the linear mean stress model, which, in the mathematical formula, has no an advantage over the Marin's mean stress model, presented in this paper? The reason is on basis of the following two aspects:

- A. By using the Marin's mean stress model, the author ever attempted to propose a multiaxial fatigue limit model, and found that numerical computation difficulty occurred sometimes in determining material constants in the Marin's mean stress model.

When non-proportional loading fatigue is carried out, Itoh's formula¹⁹ is usually used to reveal the relationship of the amplitude of the equivalent non-proportional stress, σ_{ean} , and the amplitude of the effective proportional stress, σ_{ea} , i.e.,

$$\sigma_{ean} = \sigma_{ea} (1 + \beta F) \quad (24)$$

where β is material sensitivity parameter to load-path non-proportionality defined on $\sigma - \sqrt{3}\tau$ stress plane; F is usually called a non-proportional loading factor which expresses the severity of non proportional loading. The author finds that the prediction results of fatigue life obtained by using Itoh's formula sometimes are not satisfactory. Thus the following attempt is made:

$$\sigma_{ean} / \sigma_{ea} = \lambda + \gamma F \quad (25)$$

where λ , γ are constants to be determined by linear fitting of σ_{ean} and F . Obviously, the Itoh's formula (24) has mechanical advantage over the linear formula (25). But verification by experimental data of fatigue life of metallic materials shows that the accuracy of fatigue lives predicted by means of the linear formula (25) is higher than that by the Itoh's formula (24), which will be reported in another paper.

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None.

Conflicts of interest

The authors declare that there is no conflict of interest.

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