

# Numerical and experimental investigation on low cost solar water heater for rural areas of oman

## Abstract

Solar energy is an alternative source of renewable energy. Conversion of solar radiation into useful heat is the simplest applications of solar energy, which can be used in water heating systems effectively. The most common used device to convert the solar radiation into heat is the flat plate collector. A lot of studies have been conducted to design and analyze the flat plate collector's thermal behavior and operations and to improve its efficiency. In this work, computational fluid dynamics and finite element method analysis on a flat plate collector were carried out to design and fabricate the low cost solar water heater for rural areas of Oman. The developed prototype delivered 100 litres of hot water with an average temperature of 50°C at 6 PM, which is considered as acceptable conditions for smaller families in Oman. Experiments were carried out at different conditions to validate the proposed design and obtained satisfactory results. The experimental results confirmed the validity of boundary conditions used for FEM and CFD analysis. The water outlet temperature of 83°C was observed in the afternoon session and this temperature would be ideal for domestic applications in Oman.

**Keywords:** solar energy, flat plate collector, fem, cf, oman

## Research Article

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## Introduction

Today's world is facing many problems with its limited non-renewable energy and global warming. In the last few years (1970-2013) the emission of carbon dioxide in Oman has substantially increased due to the usage of liquid fossils and natural gas.<sup>1</sup> Therefore more attention has been focused on another source of energy which is the renewable energy such as solar and wind. In this aspect, Sultanate of Oman government showed initiation on the utilization of solar and wind energy for power generation and domestic applications.<sup>2,3</sup>

Sultanate of Oman geographical location on the equator helps to develop a hot climate with 300 sunshine days per year make it an ideal location for exploring and utilization of solar energy.<sup>3-5</sup> In Oman, some companies can provide devices for houses either for electricity and water heating. However, they don't offer it cheaply and their price not affordable to every Omanis.<sup>6-8</sup> Hence, to utilize a significant amount of unused solar energy and also to save the environment from fossil fuel emissions, the proposed work signifies the development of cost-effective solar water heater for rural areas of Oman. Several researchers studied the performance of solar water heaters, and few are discussed in the present work.

Dharmasa<sup>3</sup> conducted a cost effective analysis of solar and wind power in Oman concentrating on Photo Voltaic (PV) and Wind Turbine (WT) systems. It was found that combined operation of solar PV and WT systems provided the optimal cost-effectiveness and feasibility.

Jha<sup>6</sup> analyzed the estimation and cost of solar PV system and its application in water pumping for a remotely located community in Oman. It was found that initial price of the PV system was higher than a diesel engine. Al-Badi & Albadi<sup>7</sup> presented a case study on economic feasibility of installing solar water heater technologies (SWHs) in residential units of Oman. It was found that Oman weather has ideal conditions for installation of SHWs and by using SHWs Oman can

save up to 1859 GWh annually. The installation cost of SHWs for the typical residential unit was around 10-year payback period and also, recommended that government should provide subsidies.

Gastli & Charabi<sup>8</sup> conducted a case study on solar water heater technologies (SWHs) for saving energy in Oman using RET Screen software. It was found that in Seeb district of Oman annual energy saving is around 335,431 MWh and the study results inferred that future industrial applications do not need the construction of additional power plants. The study results highlighted that the government must provide 50% sharing of the capital cost to use SWHs in their households. Chaichan<sup>2</sup> carried out experiments on in-house developed solar desalination system for Omani families. It produced one to four liters of distilled water for a day. This experiment was part of a program done by Omani Society to raise the importance of using renewable energies in the public and private sectors.

Alvarez<sup>9</sup> presented a mathematical model of flat plate collector to study the thermal and hydrodynamic behavior of the collector. 2D and 3D-hydrodynamic and thermal simulation were carried out to determine the thermal efficiency, maximum surface temperature, overall heat loss coefficient and distribution of temperature around the collector. FEM analysis results were validated by experiment. The observed temperature rise of 18.7°C at zero tilt angles and an efficiency of 84.5% was obtained. Also, experimental results highlighted that useful energy transferred to the fluid is 45 W/m at 12:30 PM and minimum energy transferred to the fluid is 1 W/m at 5:00 AM. Kazeminejad<sup>10</sup> conducted a numerical analysis of parallel flow-flat-plate solar collector with steady-state conduction equation to observe the temperature distribution above the absorber plate. The results revealed that many parameters could affect the performance of flat plate collector, the parameters can be specified as operational, meteorological, environmental parameters; the fluid inlet temperature, tube spacing, and mass flow rate strongly affect the performance of

the collector. Rama<sup>11</sup> carried out numerical simulations to describe the performance of solar collector; study indicated that two-dimensional model is an excellent tool to optimize the design of flat plate collectors, also mention that flat plate collectors can be designed for any applications which required energy delivery up to 100°C. Experiments were carried out at different localities to validate FEM results. The study results revealed that efficiency at the beginning is increased, and decreased later due to the increase in the ratio of tube spacing to the outer diameter.

Alvarez<sup>9</sup> conducted thermal and hydrodynamic modeling of a flat plate collector with the corrugated channel and validated with experiments. Their work highlighted that serpentine geometry gave better performance than parallel one. Sanjay & Dheeraj<sup>12</sup> fabricated and studied V-through flat plate collector in hot conditions.<sup>13</sup> Their work indicated with the use of low-cost materials, the developed unit could deliver 30L of water at 50°C as an average temperature in comparison to commercially available flat plate collector.<sup>14</sup> The study mentioned that reflecting mirror and change of color for the tubes would help to increase the absorption of heat and thermal efficiency with a minimum loss of energy.<sup>15</sup>

With this thought, in the present work developed a low cost solar water heater and studied the performance for Oman conditions. CFD and FEM approach was used to finalize the geometry details for the flat plate solar water heater. The proposed set-up could be affordable with lower prices for a rural community in Oman and also signified the efficient utilization of solar energy in Oman.<sup>16</sup>

## Analysis on flat plate collector

### Intensity of solar radiation

The placement of earth's orbit around the sun is such that sun-earth distance varies by 1.7 percent and solar radiation outside the earth's atmosphere is nearly of fixed intensity. The radiant energy flux received per second by a surface of unit area held normal to the direction of sun's rays at the mean earth-sun distance is constant throughout the year. This is called as solar constant,  $I_{sc}$  and its value is 1367 W/m<sup>2</sup>. However, the space radiation suffers variation due to the fact that earth rotates around the sun not in a circular orbit but follows an elliptical path. The intensity of extraterrestrial radiation  $I_{ext}$  measured on a plane normal to the radiation on  $n^{th}$  day of the year is given in terms of solar constant ( $I_{sc}$ ):<sup>17</sup>

$$I_{ext} = I_{sc} [1 + 0.033 \cos] \quad (1)$$

In addition, on the basis of transmission coefficients, latitude, declination and hour angle the total received on horizontal surface at ground is calculated from the relation

$$I_{total} = I_{ext} (\tau_b + \tau_d) (\cos\phi * \cos\delta * \cos\omega + \sin\phi * \sin\delta) \quad (2)$$

Where,  $\tau$  is transmission coefficient,  $\phi$  is latitude angle,  $\delta$  is declination and  $\omega$  is hour angle.

### Thermal analysis on flat plate collector

In the study of flat plate collector, intensity of solar radiation is represented through  $I_{total}$ ; so the total amount of solar radiation received by the flat plate collector is equal to

$$Q_i = I_{total} \times A \quad (3)$$

Where  $A$  is area of the collector in m<sup>2</sup>.

During working conditions, the collector will absorb more heat and its temperature will be higher than surrounding temperature, and heat will be lost to surrounding atmosphere by radiation and convection. The rate of heat loss ( $Q_o$ ) is depending on two factors, the overall heat transfer coefficient ( $U_L$ ) and temperature difference across the collector.<sup>18</sup>

$$Q_o = U_L A (T_c - T_a) \quad (4)$$

In addition, the rate of useful energy absorbed by the collector is equal to heat taken by water. Hence, useful energy gained by collector is explained as

$$Q_u = Q_i - Q_o = m c_f (T_c - T_i) \quad (5)$$

The performance of flat plate collector can be measured by the collector efficiency which can be defined as ratio of useful energy gain ( $Q_u$ ) to incident solar energy over a period of time

$$\eta = \frac{\int Q_u dt}{A \int I dt} \quad (6)$$

### Computational fluid dynamics

CFD analysis is a type of fluid analysis in which specific algorithm and numerical equations are used to solve problems related to fluid flow. In this work, computational fluid dynamics analysis was used to find the pressure drop at designed collectors and also to evaluate the film heat transfer coefficient,  $h$  for the flow.<sup>19</sup> In CFD, the governing partial differential equations for energy, momentum and mass were solved for steady incompressible flow.<sup>20</sup> The velocity-pressure coupling had been done by using SIMPLE algorithm (Semi Implicit Method for Pressure Linked Equation) and the mean flow equations are represented in equations (7), (8) and (9).

For Continuity equation,

$$\frac{\partial \rho}{\partial t} + \text{div}(\rho \mathbf{u}) = 0 \quad (7)$$

Momentum equation,

$$\frac{\partial(\rho \mathbf{u})}{\partial t} + \text{div}(\rho \mathbf{u} \mathbf{u}) = - \frac{\partial p}{\partial x} + \text{div}(\mu \text{grad } u) \quad (8)$$

and Energy equation,

$$\frac{\partial(\rho E)}{\partial t} + \text{div}(\rho E \mathbf{u}) = - p \text{div } \mathbf{u} + \text{div}(k \text{grad } T) + \Phi \quad (9)$$

Where  $\rho$  density of fluid,  $u$  is velocity,  $\mu$  is viscosity,  $E$  is energy,  $p$  is pressure,  $k$  is thermal conductivity and  $T$  is temperature.

Conservation equations were solved in order to yield the velocity and temperature for the fluid (water) flow in the copper tubes.

### Finite element analysis

FEM analysis for flat plate collector panel with absorption plate was studied under steady state conditions. Energy balance for the absorption plate can be calculated as

$$\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = \frac{U_L}{kt} \left[ T - T_a - \frac{I_{tot}}{U_L} \right] \quad (10)$$

Similarly, useful energy gain in the absorption plate was utilized to heat the water entering the collector, the energy balance equation on the fluid can be represented as

$$m C_f \frac{dT}{dx} + \frac{U_o}{2} (\pi D_o) (T_f - T_a) = q_u(x) 2t + I_{tot} D_o \quad (11)$$

Where  $m$  is mass flow rate of fluid,  $U_o$  is overall heat transfer coefficient,  $D_o$  is outer diameter of tube and  $t$  is thickness of tube.

Energy balance equations were solved iteratively to obtain the temperature contours across the collector.

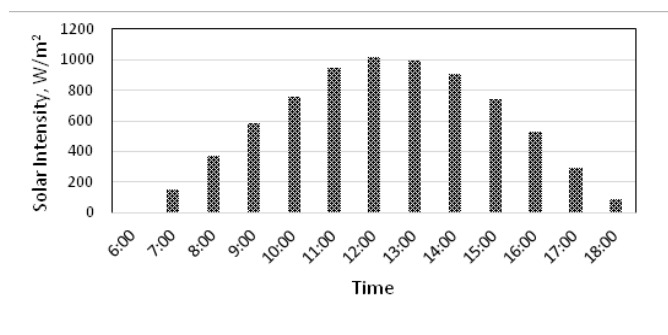
## Results and discussion

### Solar intensity of radiation

The solar intensity calculation had been carried out to determine intensity of solar radiation in Oman. The study highlighted that solar intensity in the summer period is around 1000W/m<sup>2</sup>. However, experimental study indicated a maximum solar intensity of 1017 W/m<sup>2</sup> as shown in Figure 1. In the late evening, this value is reduced to 91 W/m<sup>2</sup>. Thus, the study gave confidence in solar intensity calculations (Table 1).

**Table 1** Thermal analysis results for flat plate solar collector

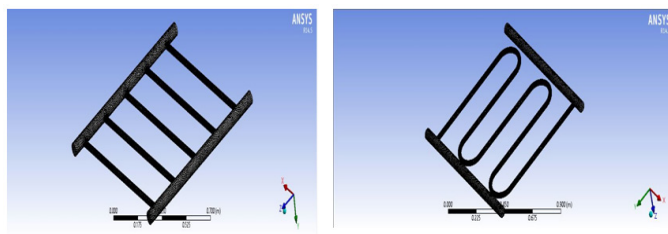
Parameter	Values
Collector heat input	1367 W
Heat loss from collector	70 W
Useful energy input to collector	1297 W
Efficiency	56%



**Figure 1** Solar intensity Data (Date: April 26, 2016).

### CFD analysis on flat plate collector

The straight tube and serpentine geometry were considered for the CFD analysis to find the pressure drop at designed collectors and heat transfer coefficient, *h* for the flow. The modeling part of fluid domain was prepared through *Pro-ECAD* software and imported to *ANSYS-FLUENT* platform. In order to predict the accurate pressure drop fine meshing approach was incorporated in the current study. The meshed details are highlighted in Table 2 and meshed geometry is presented in Figure 2.



**Figure 2** Meshed model of CFD analysis. (A) Straight tubes. (B) Serpentine.

**Boundary conditions:** The accurate boundary conditions plays an important role in CFD analysis. From the entry flow rate, the flow

velocity of 0.004 m/s was considered for the simulation and fluid temperature of 303 K was chosen for the current study as highlighted in Table 3.

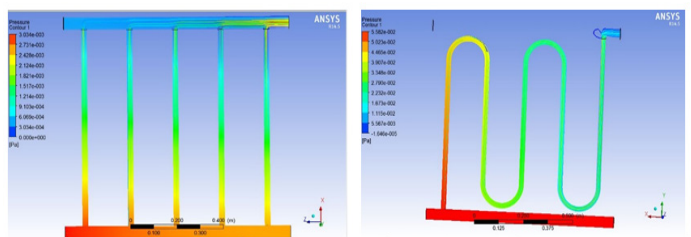
**Table 2** Mesh details – CFD analysis

	Straight pipe	Serpentine
Number of elements	355345	304315
Number of nodes	74963	66002
Mesh type	Tetrahedral – 10 node element	Tetrahedral – 10 node element

**Table 3** Boundary conditions used for the simulation

Type of geometry	Straight	Serpentine
Velocity at inlet point, m/s	0.004123	0.004123
Entry fluid temperature, K	303K	303K
Wall Material	Copper	Copper
Fluid	Water	Water

**CFD results:** Figure 3 revealed the pressure drop variation across the straight tubes and serpentine geometries. The study results indicated that in straight tubes geometry the flow enters uniformly in all the tubes and maximum pressure drop of 0.003Pa was noticed at the outlet. The low pressure drop was due to the chosen pipe dimensions of 25.4 mm diameter and 1 m length which did not caused the reverse flow in the flow path. However, serpentine geometry with 5-passes showed pressure drop value of 0.056Pa. This could be due to the fact that, in serpentine arrangement, flow enters through the zigzag path and augments the pressure drop at the outlet. Also, the study results indicated that further increase of the passes in serpentine geometry will be contributing more pressure drop across the flow. Hence, the results clearly indicated that straight tube flow path was better than serpentine geometry.



**Figure 3** Contours of pressure drop – Straight tubes and Serpentine geometry. (A) Straight tubes. (B) Serpentine.

Similarly, flow velocity contours for the straight tubes and serpentine geometry are presented in Figure 4. The results indicated that there is no reverse flow in the flow path and uniform flow without eddies was noticed with straight tubes geometry. The maximum flow velocity of 0.03 m/s was noticed with serpentine geometry as against a minimum value of 0.007 m/s with straight tubes. This could be due to the fact that the serpentine geometry will be contributing more pressure drop which indirectly augments the higher outlet velocity of the flow.

In addition, CFD analysis gave important information on average film heat transfer coefficient across the flow path. The calculated Nusselt number from the simulations indicated a heat transfer

coefficient value of 18 W/m<sup>2</sup>K with straight tubes geometry. This value was used in FEM analysis for predicting the temperature across the tube.

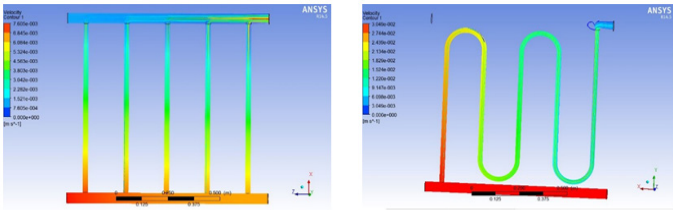


Figure 4 Velocity contours for the Straight tubes and Serpentine geometry. (A) Straight tubes. (B) Serpentine.

**Finite element analysis**

CFD analysis results revealed that straight tube geometry is found to be better for the proposed low cost solar water heating system. Hence, in FEM analysis straight tube geometry was used to simulate the temperature contours across the tube. The proposed collector assembly was modelled through *Pro-E* and imported to *Ansys-Workbench*. The meshed details are highlighted in Table 4 and the meshed geometry is indicated in Figure 5.

Table 4 Mesh details – FEM analysis

Number of nodes	19016
Number of elements	6082
Mesh type	Tetrahedral 10 node

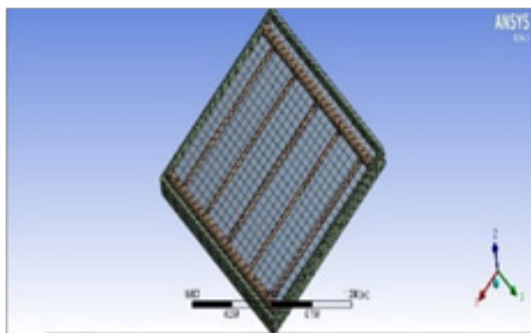


Figure 5 Flat plate collector assembly- meshed model.

**Boundary conditions:** For accurate prediction of temperature across the collector tubes, the following boundary conditions were used for FEM-thermal analysis, (Table 5).

Table 5 Boundary conditions used for FEM analysis

	Value	Zone
Heat flux	950 W/m <sup>2</sup>	Absorption plate
Film heat transfer coefficient	18W/m <sup>2</sup> K	Tubes
Ambient temperature	35°C	
Material	Copper	Tubes
Insulated condition	-	Box, Insulation plate

**FEM results:** The temperature distributions across the collector tubes is shown in Figure 6. The study results indicated 77°C - 84°C temperature variation across the collector tubes and these temperatures gave information on outlet water temperature from the solar water heater. In straight tubes geometry, all the tubes experienced uniform temperature and the maximum temperature was around 84°C at the flow exit. Thus, FEM thermal analysis coupled with CFD results indicated that five straight tubes will be sufficient to achieve the outlet water temperature of 84°C with minimum pressure drop.

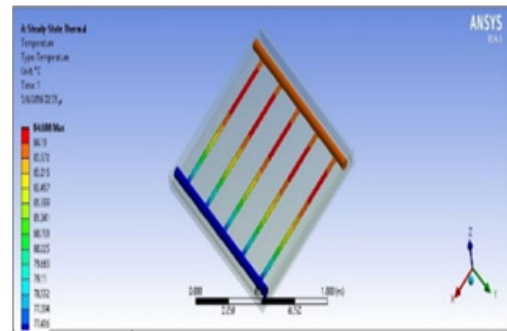


Figure 6 Temperature contours across the tubes.

**Experimental set-up:** In order to validate the CFD and FEM results, experimental set-up was developed in-house to study the performance of low cost solar water heater. In the proto model development, low cost wooden box, aluminium absorption plate and copper tubes were used. The developed low cost collector was connected to overhead water tank, which is the typical water storage arrangement in Omani houses as shown in Figure 7. In addition, the complete experiment set-up with water flow path is indicated in Figure 8.

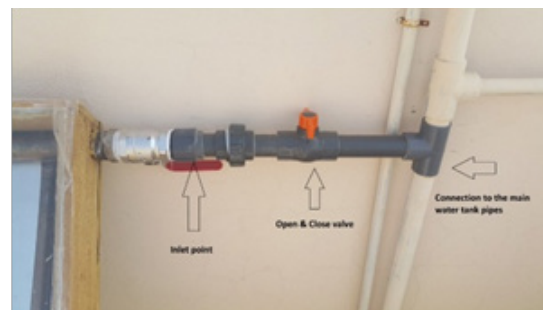


Figure 7 Pipe line path inlet to solar water heater.

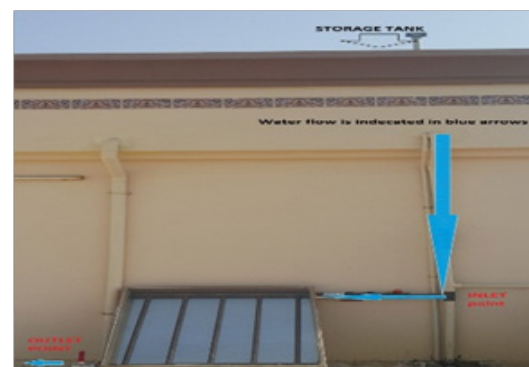


Figure 8 Experimental set-up of low cost solar water heater.

**Experimental results:** To study the performance of low cost solar water heater, experiments were carried out in the month of April, 2016. Pyranometer unit was used to record the solar intensity values and two thermometers were mounted at the inlet and outlet of collector to record the water temperatures. For each day, data were collected from 7AM to 6PM to know the behavior of the water heater. The variation of water outlet temperature at different dates is highlighted in Figure 9. On April 24, 2016 the minimum water outlet temperature observed was around 61°C. This could be due to cloudy climatic conditions which led to low solar intensity values. The experimental test results indicated maximum temperature of 83.4°C on April 27, 2016 at 12:00 PM and observed solar intensity value was around 1017W/m<sup>2</sup>. The experimentally observed outlet water temperature results almost matched with the FEM simulation value of 84.6°C as indicated in Figure 6. It gave confidence to boundary conditions used for CFD and FEM simulations.

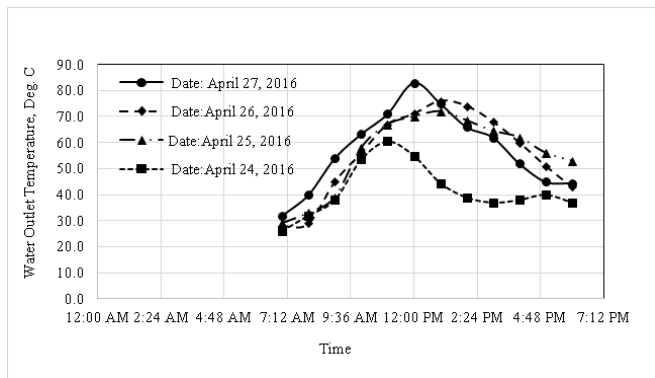


Figure 9 Variation of Water outlet temperatures.

In order to experience the seasonal conditions of sun orientation in Oman, experiments were carried out with different tilting angles of collector as represented in Figure 10. The water outlet temperature was recorded for tilting angles of 42°, 66° & 90° on the collector. From Figure 10, it is confirmed that up to 12PM, collector tilting angle of 90° was effective due to maximum concentration of solar radiation on the collector. However in the afternoon sessions around 2PM, 66° tilting angle gave better results than the 90° orientation of the collector. This could be due to orientation of the sun and amount of sun rays dropped on the collector. Thus, this study gave information about tracking of the solar rays on collector surface which would give better performance for the water heating system. However, without a tracking system, 90° tilting angle orientation for the collector was better for water heating in Oman.

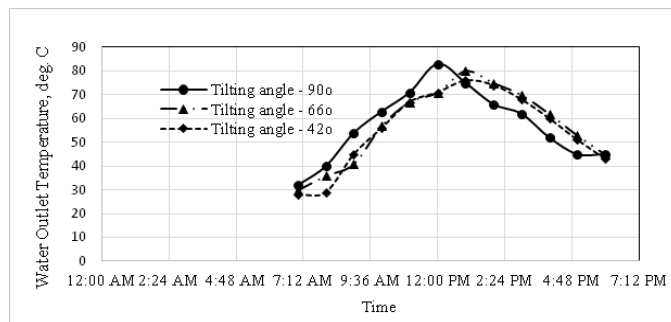


Figure 10 Variation of Water outlet temperatures with different tilting angle.

**Flat plate collector performance:** The measured experiment data were used to evaluate the energy gained by water circulating through heating system. The study showed that the collector performance is directly related to energy taken by water and absorbed solar intensity. It is observed that a collector efficiency of 50~55% for the designed low cost water heater is almost nearer to the value predicted from thermal analysis as represented in Table 6. However, on April 24, 2016, energy gained by water was less due to cloudy weather.

Table 6 Results of collector performance

Date	Max. solar intensity, W/m <sup>2</sup>	Q <sub>water</sub> , MJ	η <sub>collector</sub> , %
*April 24	850	2.62	42
*April 25	990	3.55	50
*April 26	1017	3.89	53
*April 27	1030	4.21	55

\*- Data recorded in the year 2016

### Cost comparison

The developed flat plate solar water heater consists of low cost wooden box, aluminium absorption plate, copper tubes and insulation sheets as shown in Table 7. Fabrication of low cost water heater was carried out in-house and its cost was compared with the price of commercial solar water heater available in Oman market. The cost comparison analysis showed that the cost of low cost water heater was at least 70% lower than commercially available solar water heater in Oman. Moreover, the developed low cost water heater delivered 100 litres of water with an average water temperature of 50°C.

Table 7 Low cost water heater parts and cost comparison

	Dimensions
Wooden Box	1m x 1m x 20 mm
Absorber plate	0.980m x 0.980m x 10mm
Insulation layer	0.980m x 0.980m x 20mm
Pipes (Water entry and exit)	φ50.8 mm x 0.980m
Absorbing pipes	φ25.4 mm x 0.980m
Cost of Low cost of water heater = OMR 90	
Cost of Commercial water heater = OMR 160	

### Conclusion

In the present work, performance of low cost flat plate solar water was studied for Oman conditions. A proto model was developed in-house and produced 100 litres of hot water, which is considered as normal consumption of hot water for smaller houses in Oman. CFD and FEM approach was used to finalize the dimensions for solar water heater. The following are the outcomes of the present work:

- i. CFD results indicated that straight absorber tubes geometry is preferable than serpentine configuration.
- ii. The experimental results highlighted a maximum water temperature of 83.4°C, which is almost in line with FEM simulated values under steady state conditions.
- iii. Tilting angle of 90° is better until 12PM and for afternoon sessions

66° tilting angle showed good results due to orientation of solar radiations on the collector surface. However, in the present set-up solar tracking system was not incorporated and from low cost point of view, 90° tilting angle would be an ideal option for Oman conditions.

- iv. Moreover, study results showed the collector efficiency of 50~55%, which is almost equal to efficiency obtained through thermal calculations.
- v. In addition, cost of the developed water heater is at least 70% lower than the commercially available water heater in the market.

## Acknowledgements

None.

## Conflict of interest

The authors declare no conflict of interest.

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